

# Flow Through a de Laval Nozzle

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## 1. Overview of the case

In this project, the canonical problem of a de Laval nozzle is simulated under compressible conditions using OpenFOAM.

De Laval nozzles use changes in area in an interior flow setting to transform a subsonic flow into a supersonic flow. They consist of three regions: an inlet, a narrow throat, and an outlet. If set up correctly, flow through a de Laval will begin as a subsonic flow, transition between subsonic and supersonic at the throat, and remain supersonic downwind of the throat passing through the outlet. The flow itself is driven by a pressure gradient along the nozzle.

The de Laval simulated here consists of interior flow through a cylinder whose radius is a smooth function with  $z$  (defined in the typical cylindrical coordinates as the distance along the axis of symmetry). The function of  $z$  describing the radius of the cylinder, and thus the area is chosen such that it contains a single minima within the cylinder. This forms the throat between two wider inlet and outlet regions.

## 2. Hypothesis

- Compressible Flow
- Laminar Flow
- Viscous flow
- Cylindrical Geometry ( $\frac{\partial}{\partial \phi} = 0$ )
- Negligible gravitational effects
- Steady flow ( $\frac{\partial}{\partial t} = 0$ )

- Subsonic and supersonic components

### 3. Physics of the Problem

The de Laval nozzle transformed a subsonic flow through a pipe into a supersonic flow by varying the cross-section of that pipe. Ignoring the transverse components of the flow, the derivation for this process arises as follows.

In the derivative form, the conservation equation takes the form

$$\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho u) = 0.$$

If we assume that the flow is in steady state, this reduces to

$$\nabla \cdot (\rho u) = 0$$

which under our assumption of ignoring any changes perpendicular to the  $z$  axis can be written as

$$\frac{1}{\rho} \frac{d\rho}{dx} + \frac{1}{u} \frac{du}{dx} + \frac{1}{A} \frac{dA}{dx} = 0$$

where  $A$  is the cross sectional area of the tube at a given value of  $z$  arising from scale factors hidden in the divergence operator. If we assume that gravity is negligible, ignore viscosity, and use an adiabatic equation of state, the momentum equation takes the form of

$$u \frac{du}{dx} = -\frac{c_s^2}{\rho} \frac{d\rho}{dx}$$

We can then use our version of the continuity equation to replace  $\frac{1}{\rho} \frac{d\rho}{dx}$ . This yields an equation that relates the velocity and the local Mach number of the flow in terms of the cross sectional area of the nozzle:

$$(1 - M^2) \frac{1}{v} \frac{dv}{dx} = -\frac{1}{A} \frac{dA}{dx}.$$

By examining the signs of the terms attached to each side of this equation we can gain insight about the different regimes. When the flow is subsonic ( $M < 1$ ),  $\frac{du}{dx}$  and  $\frac{dA}{dx}$  will have opposite signs. This means that as the area increases, flow speed decreases. In the case of supersonic flow ( $M > 1$ ),  $\frac{du}{dx}$  will have the same sign as  $\frac{dA}{dx}$ , meaning that as the area of the nozzle increases, velocity also increases. This is somewhat unintuitive, but turns out to be a very useful property. In the case where  $M = 1$ ,  $\frac{dA}{dx} = 0$ ; transitions from subsonic flow to supersonic flow are required to occur at minimums in the cross sectional areas of the pipe. This condition is known as ‘choked flow’.

If we construct our nozzle to have a single minima in the cross sectional area and have a high enough pressure gradient across the pipe to reach supersonic speeds at the throat, then we can accelerate a supersonic flow simply by increasing the area on the other side of the pipe. This system is known as a de Laval nozzle, and has both engineering applications (e.g. rockets) and astrophysical implications (e.g. jets from active galactic nuclei jets). This derivation and discussion is adapted from Choudhuri (1998).

The differential equations above can be solved and written in terms of  $M$  for  $A$ ,  $T$ , and  $p$  and the adiabatic index  $\gamma$ . These solutions hold true for both the subsonic and supersonic cases.

$$\begin{aligned}\frac{A}{A^*} &= \frac{1}{M} \left( \frac{1 + \frac{\gamma-1}{2} M^2}{\frac{\gamma+1}{2}} \right)^{\frac{\gamma+1}{2(\gamma-1)}} \\ \frac{T_0}{T} &= 1 + \frac{\gamma-1}{2} M^2 \\ \frac{p_0}{p} &= \left( 1 + \frac{\gamma-1}{2} M^2 \right)^{\frac{\gamma}{\gamma-1}}\end{aligned}$$

Quantities with the subscript ‘0’ are those at the inlet and  $A^*$  is the cross-sectional area at the throat of the nozzle. These equations will be useful for testing the simulations presented here. Note that this derivation ignores viscosity, so some difference is expected between the simulation results and the analytic expressions.

## 4. Pre-processing

### 4.1. Mesh Generation

The mesh for this case was generated using a custom Python script, `makewedge.py` that interacts with Gmsh, a three-dimensional finite element mesh generator. The Python script generates a wedge of an axisymmetric tube whose radius is a function of  $z$ . The generated mesh is a structured mesh comprised of boxes. The lines in the mesh that connect the axis to the outer wall take advantage of the ‘transfinite’ scaling available in Gmsh to increase resolution near the outer wall of the pipe. The script that generates this mesh is flexible enough to generate any axisymmetric tube whose radii can be written as a well behaved function. The degree of resolution in the interior of the tube and along the tube axis is also highly customizable.

Using this python script and Gmsh, a nozzle whose radii takes the form of

$$r = 1 - 0.868z^2 + 0.432z^3$$

with  $z$  values from 0 to 2 was constructed. This function was then reversed along the  $z$  axis in order to form a longer outlet than the inlet. The final form of  $r$  vs.  $z$  is shown in Figure 1. The nozzle was constructed with 400 cells along the axis, and 200 cells between the axis and the wall of the nozzle created with a transfinite progressive scaling of 0.97.

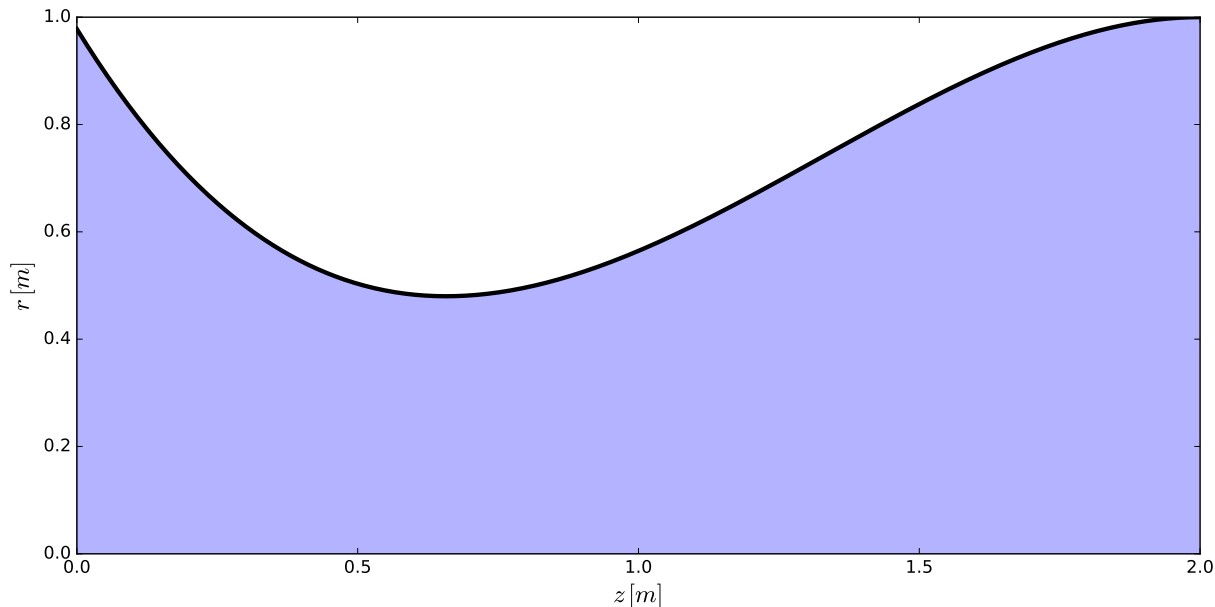


Fig. 1.—: Radius as a function of distance along the axis,  $z$ . The minima where the flow will pass from subsonic to supersonic occurs at  $z = 0.658$ .

In order to reduce computational time and simplify the meshing process, the symmetry of the system was leveraged to reduce the problem to a simple wedge rather than a full cylinder. The wedge used in this set of simulations has an angular extent of  $2.5^\circ$ . The angular extent of the mesh is a single cell wide as required by the wedge boundary conditions used by OpenFOAM. The Python script defines 7 physical regions that will later be used to set boundary conditions. These consist of the outer wall, the two faces of the wedge, the inlet, the outlet, the interior volume of the nozzle, and finally the line along the axis. A mesh described by the same function with the resolution reduced by a factor of 10 for clarity of that used in the actual simulation is shown in Figure 2. A zoomed-in version of the actual mesh is shown in Figure 3 to show the higher grid resolution near the nozzle walls. A reduced resolution version of what the entire nozzle would look like if the wedges were rotated around the  $z$  axis is shown in Figure 4

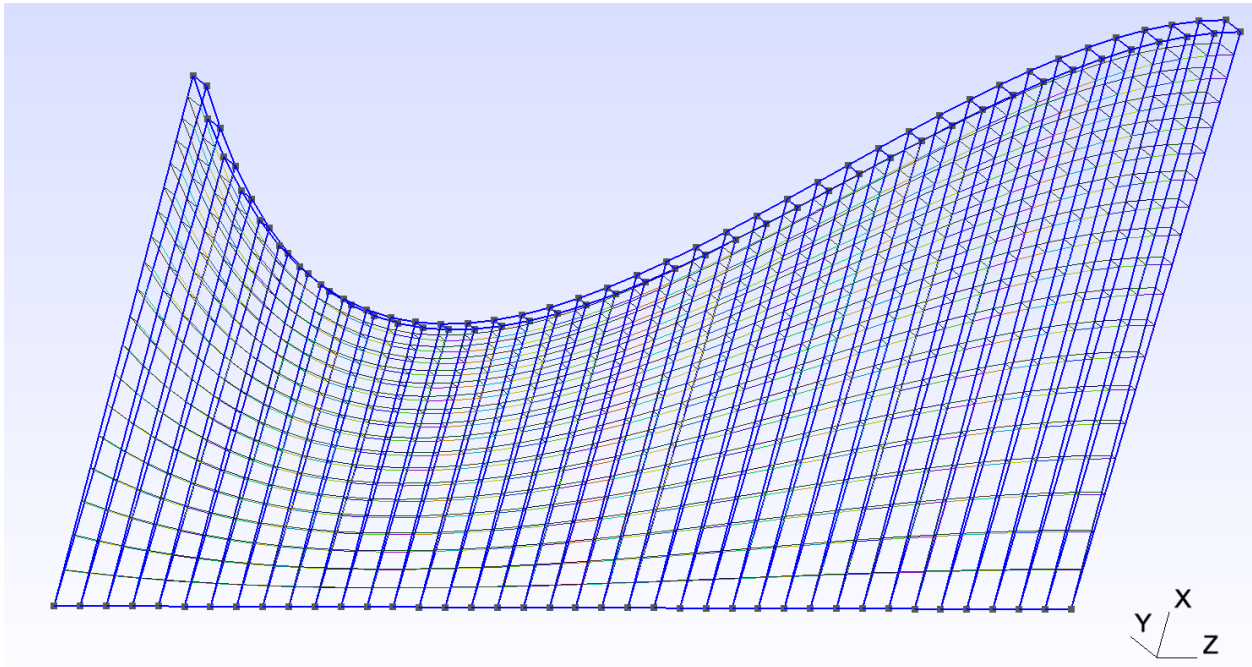


Fig. 2.—: Lower resolution representation of the mesh produced by the custom Python script and Gmsh for visualization. The mesh resolution shown here has been lowered by a factor of 10.

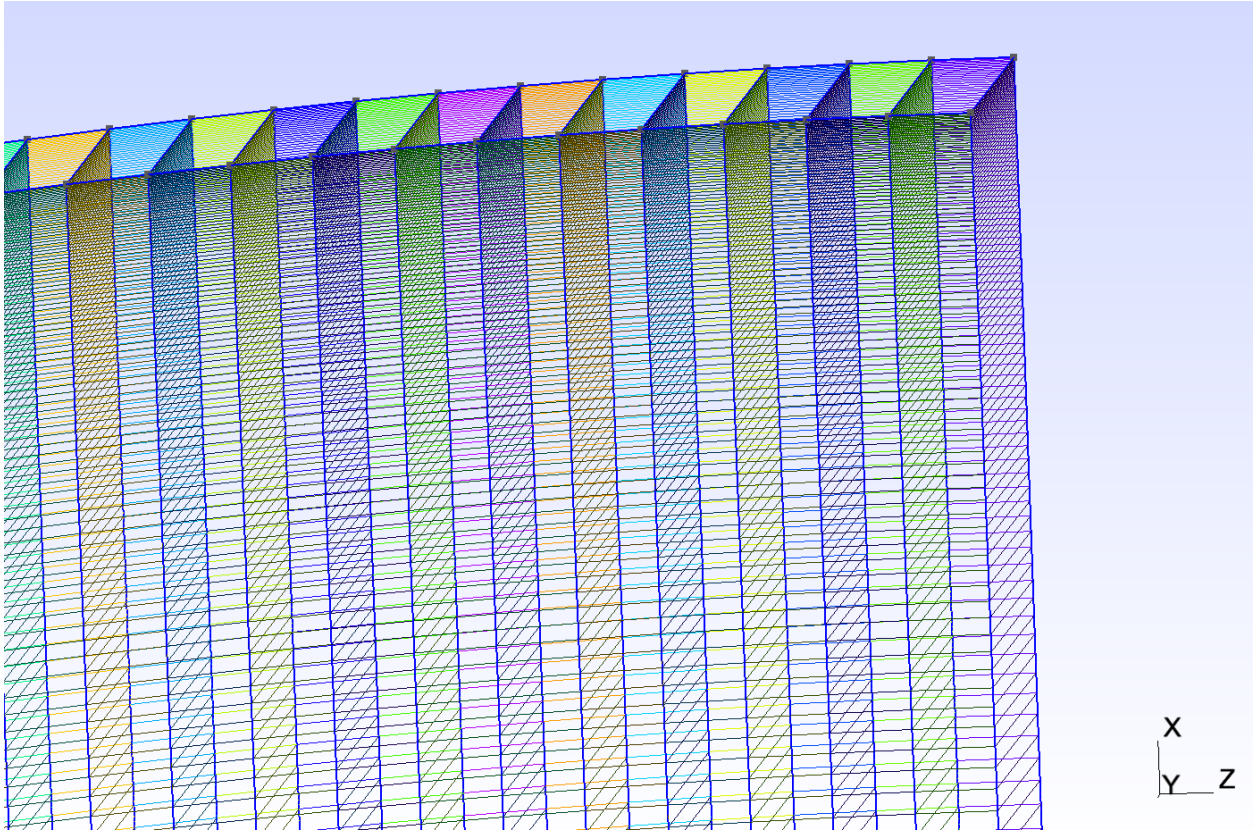


Fig. 3.—: Zoomed in view of the actual mesh used for simulation at the wall near the outlet. The resolution of the mesh increases as a function of radius to ensure that more complicated interactions with the wall are captured correctly.



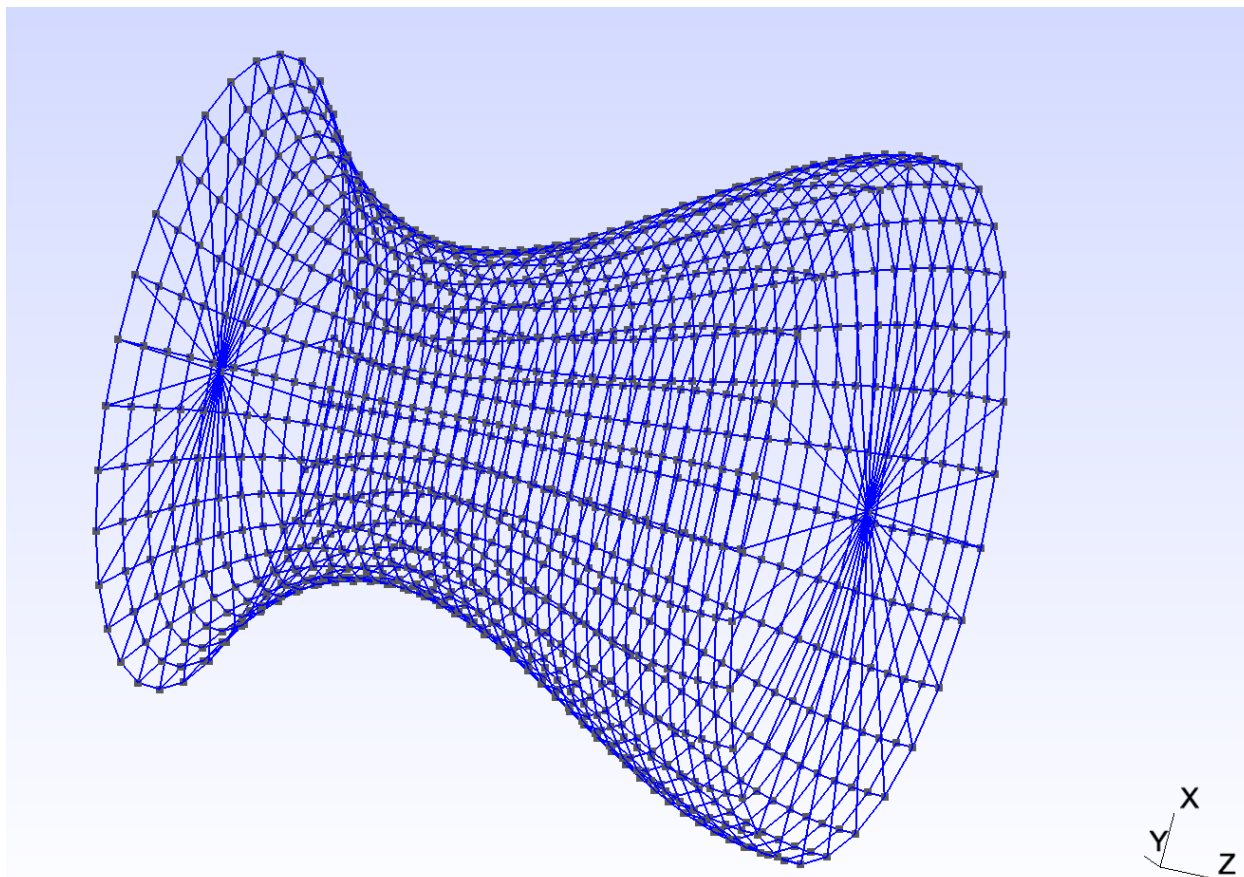


Fig. 4.—: Reduced resolution mesh of the entire nozzle rather than just a wedge.

The file that the Python script produces was then loaded into Gmsh and the mesh was then constructed using the standard 3D meshing algorithm. The mesh file produced by Gmsh was then transferred to the polymesh directory in the constant directory and converted to a format that OpenFOAM can read using the gmshToFoam command.

The python script that generates the mesh is shown in the appendix. The mesh file itself is too cumbersome to include here, but it can be easily constructed using this script using the parameters specified above. As mentioned above, the script is quite general and could be useful for many problems that allow for cylindrical symmetry. Due to

computational time constraints, only one mesh was run.

Initially, an unstructured mesh of the entire 3D nozzle was tried rather than a simple wedge, but ultimately failed. After a few time steps the CFL number became larger than 1 and the simulation crashed. This caused the movement towards the simpler wedge mesh, which allowed for the construction of a structured mesh, and reduced computing costs. Under the full 3D nozzle construction there were multiple issues with cell skewness and high aspect ratio cells that were flagged by the OpenFOAM command `checkMesh` as issues. With the simpler wedge construction, there were much fewer issues the mesh.

## 4.2. Numerical Solver

The OpenFOAM numerical solver that was chosen is `rhoCentralFoam`. This solver has the ability to handle supersonic compressible flows, which is necessary for handling the conditions expected to present within the nozzle. This scheme is a density-based compressible flow solver based on central-upwind schemes of (Kurganov and Tadmor 2000). The fluid parameters tracked throughout the simulation are temperature, velocity, density and pressure. The parameter files for `fvSchemes` and `fvSolution` are shown in the appendix.

Because this simulation is run under non-isothermal conditions, a `thermophysicalProperties` file is required. The solver uses a `hePsiTherm` `thermoType` keyword that constructs a `psiThermo` thermodynamical model based on compressibility of the fluid for a system with a fixed composition. A second keyword, `pureMixture` ensures that the simulation is run without reactions and a perfect gas is assumed. The thermodynamic qualities of the fluid are set such to approximate the fluid as air. The flow is assumed to be laminar, which is set in the `turbulenceProperties` file.

### 4.3. Boundary and Initial Conditions

The boundary conditions for this problem were defined using the physical regions defined by Python script. The line that traces the axis of the simulation uses the `empty` boundary condition and the two faces of the wedge use the `wedge` boundary condition for all parameters, making the assumption of cylindrical symmetry. The outer wall of the simulation uses a `slip` boundary condition for simplicity as the boundary layer at the wall is not the focus of this simulation. The parameters that require boundary conditions are velocity, pressure and temperature. The individual boundaries for the inlet and outlet for each parameter are described below. The parameter files for each set of boundary conditions are shown in the appendix. Because the `wedge` boundary condition was used at the two faces, the `boundary` file within the `polymesh` directory also needed to be updated.

#### 4.3.1. Pressure

The inlet boundary uses the `totalpressure` boundary which is useful for inlets where the velocity is not known a priori. This condition sets the pressure equal to a constant for outflow (a standard outflow boundary), but allows it to find its own value for inflow. The outlet boundary uses the `waveTransmissive` boundary to avoid spurious wave reflections off of the boundary and define the pressure difference across the nozzle. The choice of the pressure difference, along with the cross sectional area of the nozzle, between the inlet and the outlet is what sets the velocity and temperature distribution within the nozzle.

#### 4.3.2. Velocity

The inlet and outlet both use a `zeroGradient` boundary condition. This allows the velocity distribution to be extrapolated to the boundaries from the conditions within the

nozzle, which in the most part are governed by the difference in pressure accross the nozzle.

### 4.3.3. *Temperature*

The inlet boundary uses a `fixedValue` condition, fixed at a temperature of 298K (room temperature). The outlet boundary uses a `zeroGradient` boundary condition under the same premise as the velocity; the flow is allowed to evolve naturally under a fixed pressure gradient.

## 4.4. Control

The time step for this simulation was chosen dynamically using the `adjustTimeStep` keyword within the control file. This allows to simulation to increase the time step until the Courant number reaches a specified limit. In this simulation, a maximum Courant number of 0.5 was chosen to ensure stability. At most points during the simulation, the time step is around  $1 \times 10^{-8}$  s. The simulation was run for 0.02 seconds and the parameters with reported every  $1 \times 10^{-4}$  seconds, yielding a total of 200 snapshots.

## 4.5. Sampling

A sample of the simulation was excised along the length of the nozzle slightly above the symmetry axis. The first point in cartensian coordinates lies at (0.1 0.0 0.005) and the final point lies at (0.1 0.0 1.995). The points were sampled in evenly spaced increments along a straight line with 100 points. The ‘sampleDict’ file is shown in the appendix.

## 5. Results

### 5.1. Supersonic de Laval Nozzle

Here the results from the supersonic simulation are presented and discussed. Because of high numerical costs, only one simulation is presented. This simulation took  $\sim 4$  days to run on the Shared Computer Cluster. The solver used for this simulation is not currently parallelized, but does have the potential to be.

#### *5.1.1. Transient behavior*

As the simulation begins, shocks propagate from the high pressure inlet region and travel outward (see Figure 5). Waves reflect off of the outer walls. Due to symmetry, these reflected waves interact with their complement waves near the center of the nozzle (see Figure 6), providing a nice check that the wedge boundary conditions are working correctly. Gradually the simulation begins to settle into a steady state flow, roughly around 0.01 s.

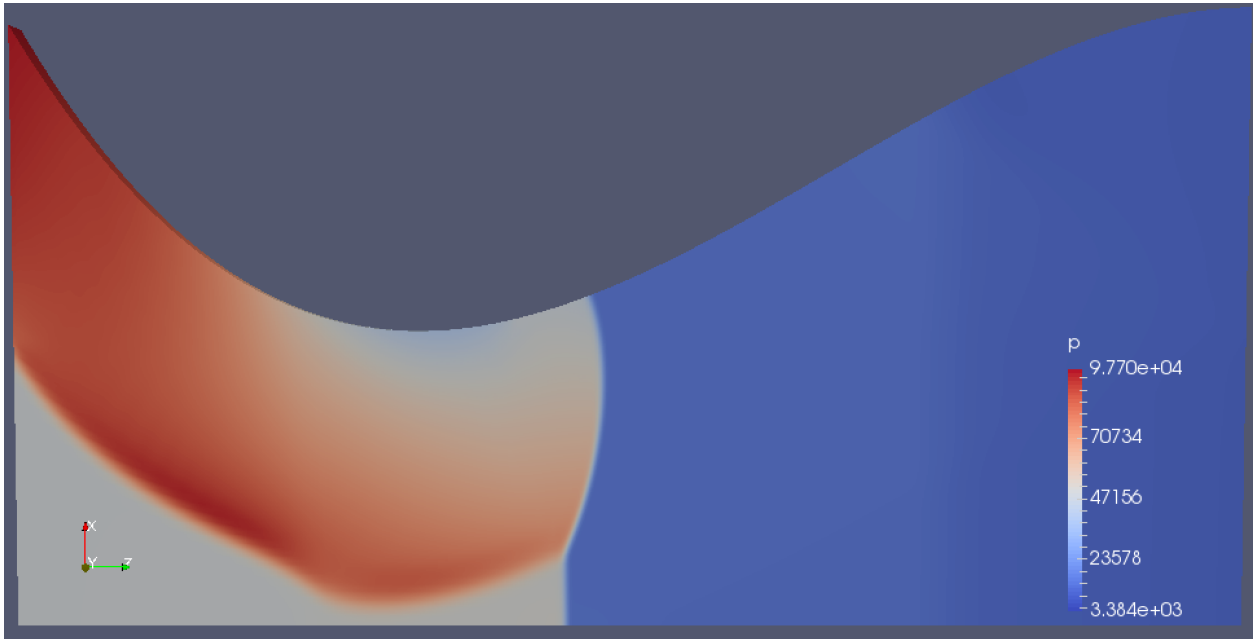


Fig. 5.—: Transient behavior shown in pressure from initialization traveling from the outlet into the rest of the simulation. Shocks are present at this point in the simulation. This snapshot was taken at  $t = 0.0007$  s.

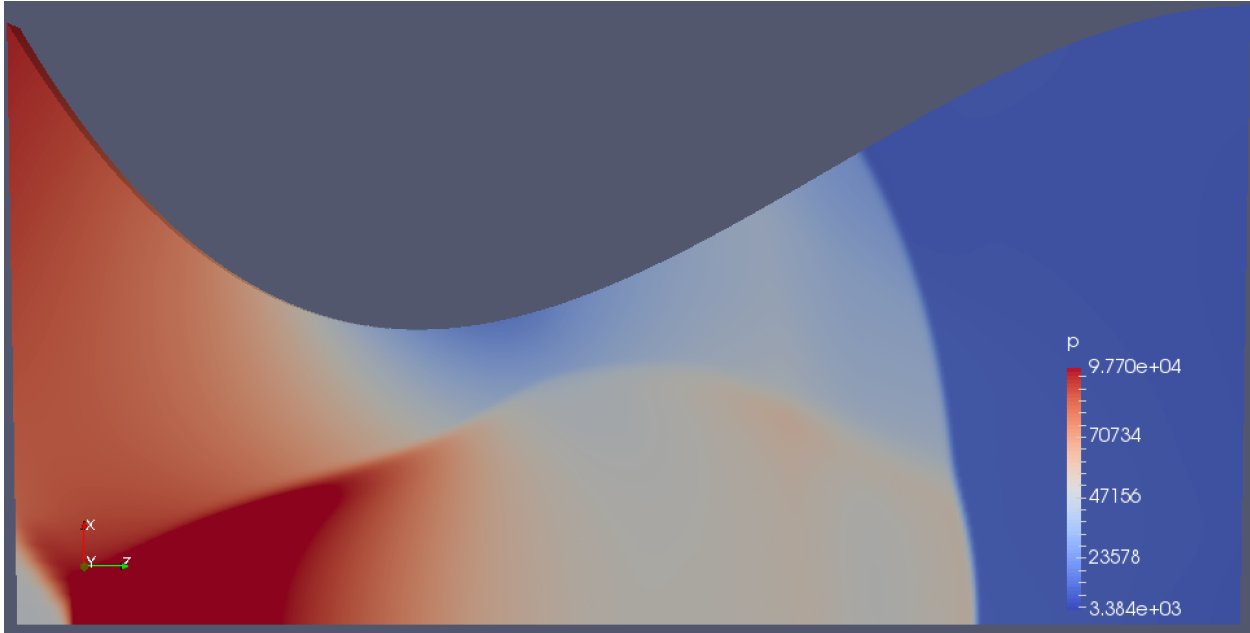


Fig. 6.—: Shocks crashing into each other along the symmetry axis. This snapshot is at  $t = 0.00115$  s. The shock front is still present at this time but is rapidly approaching the outlet.

### 5.1.2. Flow structure

Once the transient behavior has subsided, clear patterns emerge. The flow smoothly becomes supersonic roughly where the throat of the nozzle is at its narrowest point (Figure 7). Note that the line separating subsonic from supersonic flow is not quite straight, likely showing the effects of adding viscosity to the fluid. Figures, 8, 9, and 10 show the density, pressure, temperature and speed distributions across the nozzle once transients have dissipated. All of these structures presented in these figures are reminiscent of the theoretical structure predicted by the analytic inviscid case.

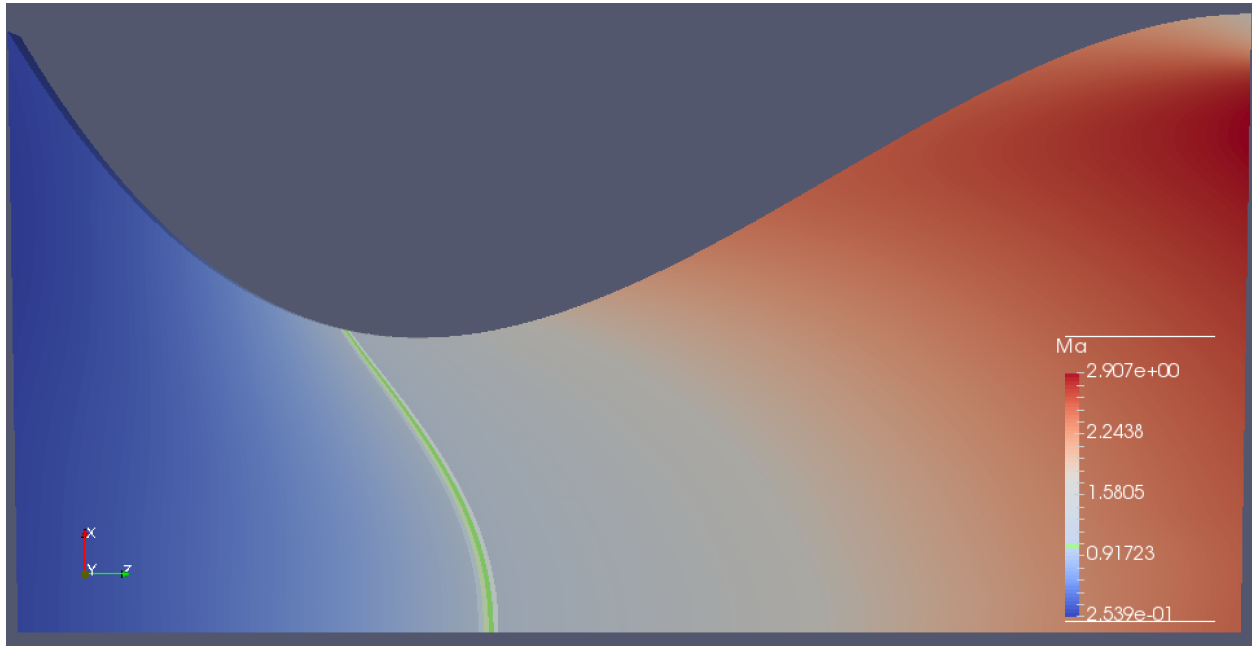


Fig. 7.—: Mach number once transients have dissipated. The green band shows where the flow becomes supersonic, roughly where  $\frac{dA}{dz} = 0$ . Note that the curved nature of the transition is not predicted by the analytic solution. Note that the Mach number contains all components of the flow, which may be part of why the sonic point is curved.



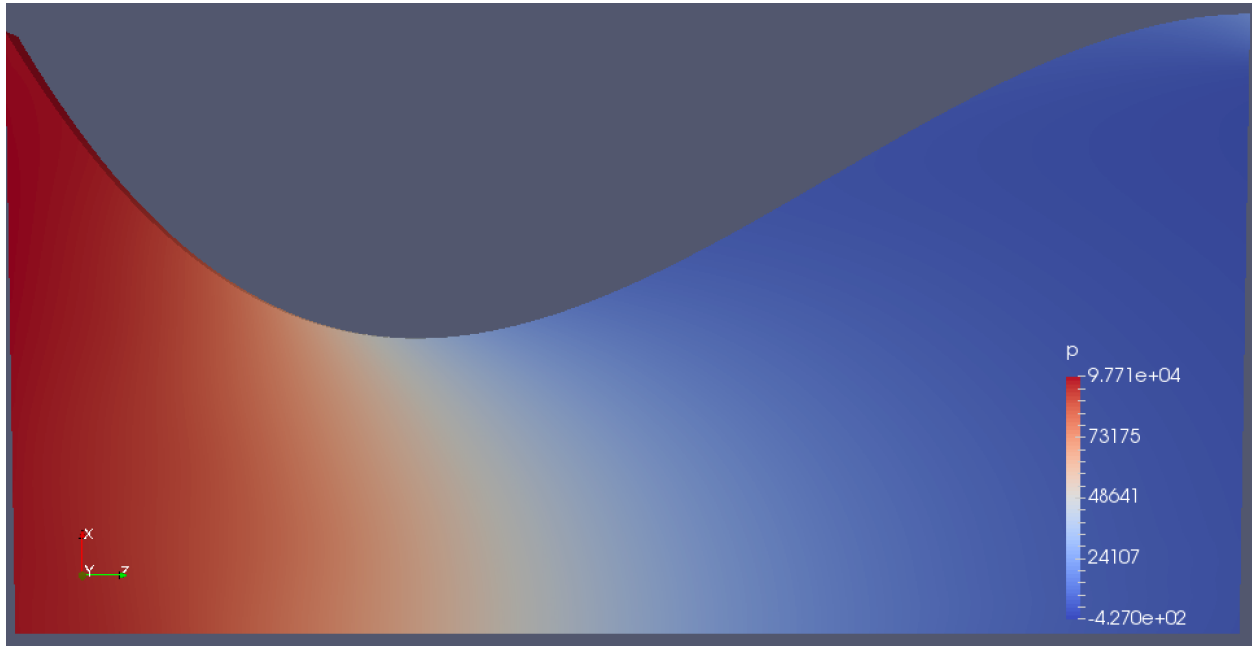


Fig. 8.—: Pressure distribution across the nozzle in  $\text{N/m}^2$ .

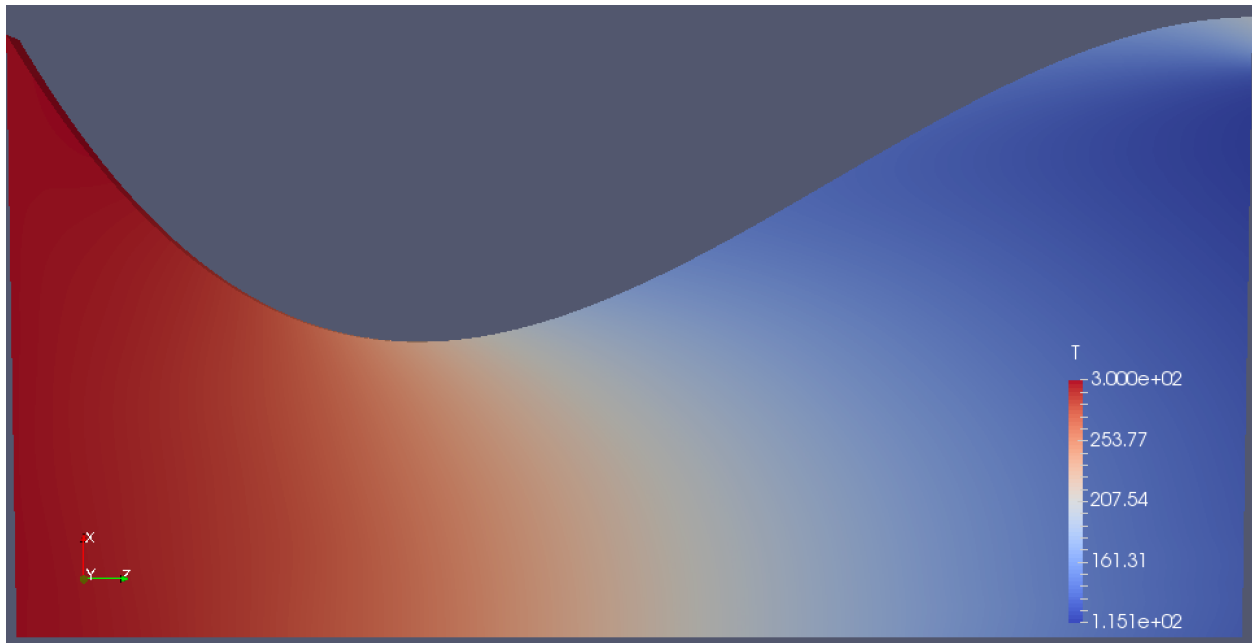


Fig. 9.—: Temperature distribution across the nozzle in K.

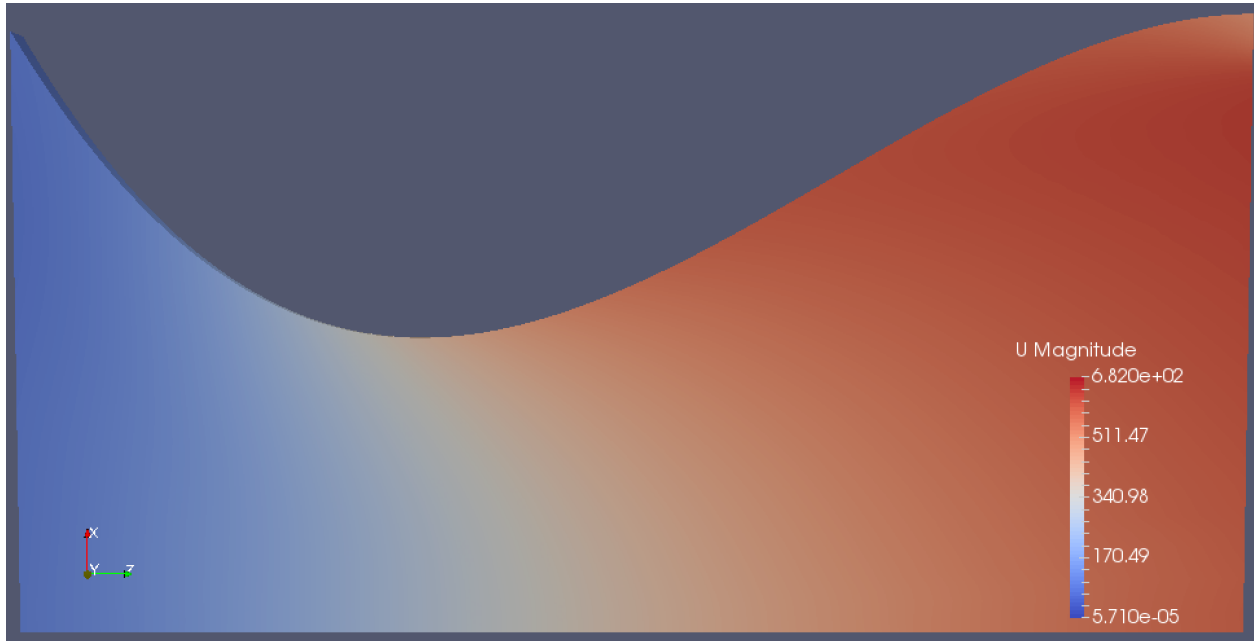


Fig. 10.—: Speed distribution across the nozzle in m/s. Note that the speed shown here includes all components of the velocity.

Figure 11 shows the mach number in the final snapshot of the simulation measured along the central axis once the flow has converged to steady state. Plotted alongside the result is the analytic result for the Mach number of the inviscid de Laval nozzle with the same geometry. Here the simulation does differ a bit from the analytic solution. The addition of viscosity, and potentially numerical artefacts from the process of simulation, have slowed down the supersonic flow, resulting in an exit Mach number of  $\sim 2.5$  rather than the predicted  $\sim 3.0$ . The flow also passes through the sonic point slightly further down the nozzle than predicted by the analytic results.

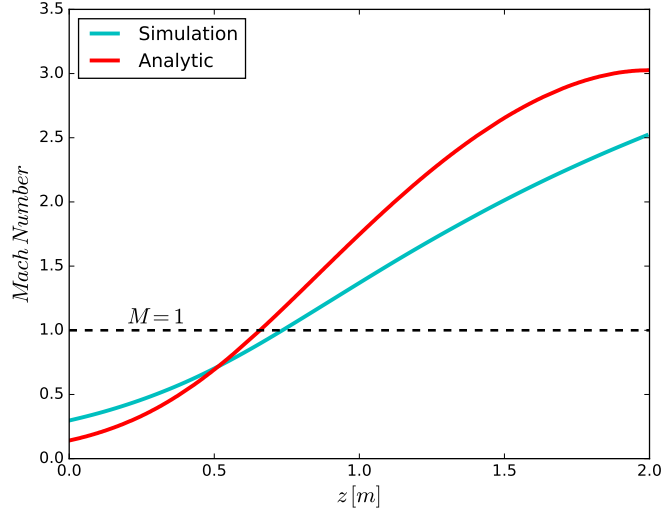


Fig. 11.—: Comparison of the Mach number as a function of  $z$  between the simulation and the analytic result for an inviscid de Laval nozzle. The sonic point is shown as dashed line. The Mach number for the simulation was sampled along the  $z$  axis at a height of 0.1m above the axis of symmetry.

A similar plot is shown for the temperature structure in Figure 12. For the subsonic flow, the temperature structure closely follows the analytic solution. However, as the flow becomes supersonic, a difference of  $\sim 40\text{K}$  gradually develops over the length of the nozzle with the simulation being hotter than the analytic solution. This is reasonable, as viscosity only present in the simulation will transform some of the kinetic energy of the flow into thermal energy. Finally, a comparison of pressure against the analytic case is shown in Figure 13. Again, in the subsonic regime before the choke point there is good agreement with the analytic case and as the flow becomes faster, larger deviations appear. Overall, the predictions made by theory seems to be fairly well replicated by the simulation given that the analytic solution is inviscid. The viscosity of the simulation presents itself by heating and slowing down the supersonic flow.

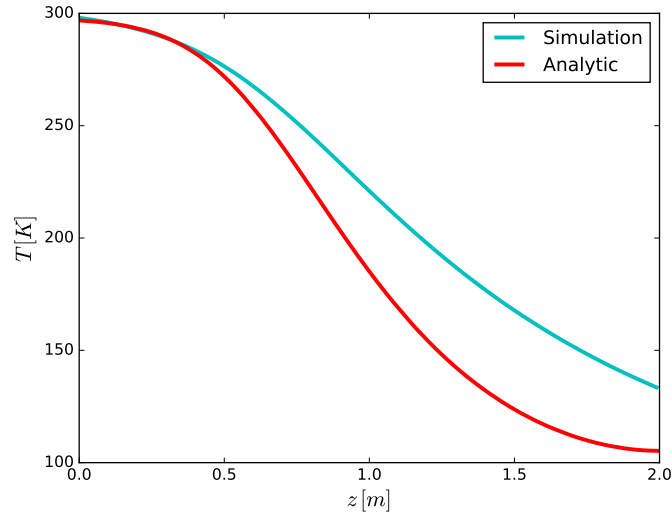


Fig. 12.—: Comparison between the simulation and the analytic solution for temperature as a function of  $z$ . The temperature of the simulation is notably higher than the temperature predicted by the analytic solution. This is likely due to the introduction of viscous forces.

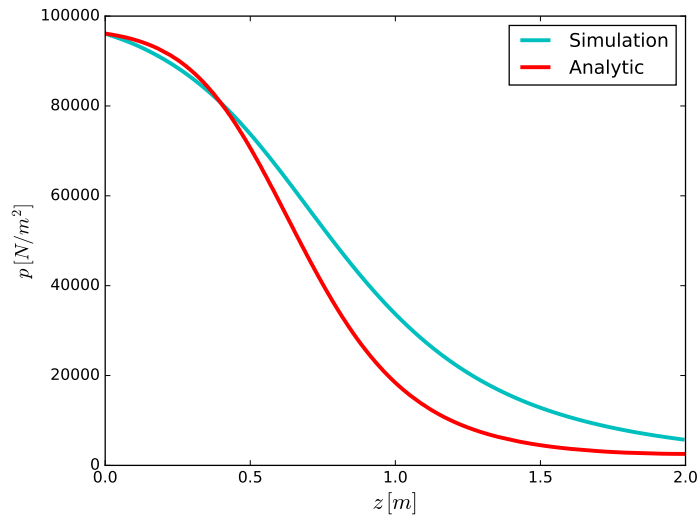


Fig. 13.—: Comparison between the simulation and the analytic solution for pressure as a function of  $z$ . Again, a deviation is seen from the analytic as the velocity is accelerated through the nozzle.

## 5.2. Convergence

When the simulation converges to steady state, the difference in tracked parameters between one time step to another should become smaller. These differences between the two steps are known as the residuals. If the residuals are plotted as a function of time one can observe how quickly the simulation approaches steady state and when it has sufficiently converged. Unfortunately, the majority of the residuals for the supersonic case were overwritten as the simulation was interrupted several times throughout its run. However, the residuals for  $U_z$  a subsonic case are shown in Figure 14.

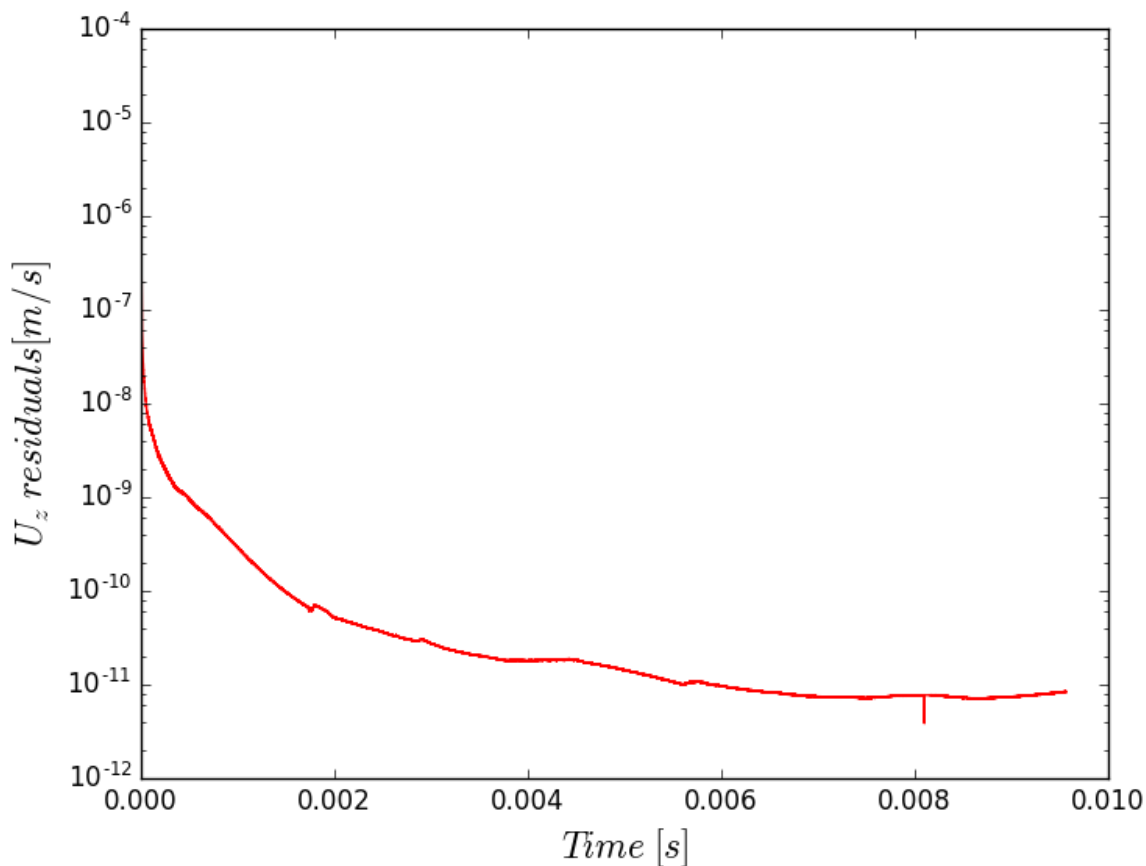


Fig. 14.—:  $U_z$  residuals as a function of time for a subsonic case. Over time the residuals decay to a roughly constant value, consistent with the simulation approaching residuals resembling numerical error.

## 6. Conclusions

A de Laval nozzle was constructed using a custom python script that generates structures meshes of wedges of axisymmetric pipes. Using the rhoCentralFoam solver packed with OpenFOAM, a steady state transonic solution was found. This solution was then compared against the known analytic result and was found to be in relatively good agreement. The

deviations from the exact solution are likely due to the addition of viscosity which would serve to slow and heat the flow as the kinetic energy is transferred into thermal energy, which is what is observed in a line measurement taken along the axis of the simulation.

Future work for this project includes investigating how the results change with grid resolution, because as it currently stands, the simulation takes a very long time to run. Because of this, it is not clear if the simulation is in a state of mesh convergence. Expanding the simulation to include the area behind the outlet would also be interesting, and would potentially allow us to study the ‘shock diamonds’ that form behind this sort of nozzle. The flexibility of the mesh generating code would make this fairly straightforward, although more physical regions would need to be defined.

## REFERENCES

- Choudhuri, A. R. (1998). *The physics of fluids and plasmas : an introduction for astrophysicists*.
- Kurganov, A. and Tadmor, E. (2000). New high-resolution central schemes for nonlinear conservation laws and convectiondiffusion equations. *Journal of Computational Physics*, 160(1):241 – 282.

## 7. Appendix

Included in this section are the parameter files that were used to run the supersonic de Laval nozzle in addition to the custom python script used to generate the structured mesh for the grid. The actual grid file is not included as it is remarkably large. The files are listed here in the order in which they are presented above.



```
import numpy as np
import matplotlib.pyplot as plt
import pdb

'''
makewedge.py

PURPOSE:
    Script that writes instructions to create a wedge shape for the CFD openfoam
    project.
    For use with Gmsh

AUTHOR:
    Connor Robinson, Dec 6th, 2016
'''

#Set up paths for saving
path = '/Users/Connor/Desktop/Coursework/CFD/project/'
outfile = 'mesh_shape.txt'

#Define the shape of the nozzle in 1D
#Set the number of cells along the cylinder
length = 400

#Set up cells between the center of the cylinder and the axis
#Specify number of cells
hnum = 200

#Specify the transfinite progressive scaling for those cells
hprog = .97

#Define r as a function of Z
zbase = np.arange(length)
r = (45-zbase**2/(.08*length)**2 + zbase**3/(.186*length)**3)
r = (r/np.max(r))[:,::-1] #Reverse r (Better nozzle shape, not necessary)

#Scale zbase back to something reasonable.
zbase = zbase/(length/2)

#Define the fraction of a circle for the wedge to include (1/144 = 2.5 degrees)
anglefrac = 1/144

#Actually go about generating the mesh, unlikely necessary to change anything below
#here.
anglepoints = 3

phi = [-np.pi*anglefrac,np.pi*anglefrac]

x = np.array([r * np.cos(p) for p in phi]).flatten()
```

```
y = np.array([r * np.sin(p) for p in phi]).flatten()

z = np.hstack([zbase, zbase, zbase])
x = np.hstack([x,np.zeros(length)])
y = np.hstack([y,np.zeros(length)])

vlines = []

#Give a number to each point
points = np.arange(length*anglepoints)+1
counter = 1

#Connect the lines along the flow axis, start at bottom and work around with
increasing phi
for angle in np.arange(anglepoints):
    for xval in np.arange(length-1):
        vlines.append([counter, xval+ angle*length+1, xval+angle*length+2])

        counter = counter +1

hlines = []
nvert = counter -1

#Connect the lines around the tube, start at bottom and work way up.
for zval in np.arange(length)+1:
    for angle in np.arange(anglepoints):

        if (zval == length) and (angle == anglepoints -2):
            #hlines.append([counter, zval+angle*length, length*anglepoints])
            hlines.append([counter, length*anglepoints, zval+angle*length])

        else:
            if angle == 2 or angle == 0:
                hlines.append([counter, zval+angle*length,
np.mod(zval+(angle+1)*length, length*anglepoints)])
            if angle == 1:
                hlines.append([counter, np.mod(zval+(angle+1)*length,
length*anglepoints), zval+angle*length])

            counter = counter+1

nhorz = counter - 1 -nvert

#Now form the line loops, start at bottom and work around with increasing phi
loop = []
loopcounter = 1

for zval in np.arange(length-1):
    for angle in np.arange(anglepoints):

        loop.append([loopcounter,\
```

```
vlines[zval+(angle*(length-1))][0], \  
hlines[(zval+1)*anglepoints + angle][0], \  
vlines[np.mod(zval + (angle+1)*(length-1), (anglepoints)*(length-1))][0], \  
hlines[zval*anglepoints + angle][0]]  
  
loopcounter = loopcounter +1  
  
#Write out the lines and line loops to a text file.  
file = open(path+outfile, 'w')  
  
#Write each set of points  
file.write('// Points \n')  
  
for p in np.arange(length*anglepoints):  
    file.write('Point ('+str(np.int(p+1))+') =  
{'+str(x[p])+','+str(y[p])+','+str(z[p])+',1};\n')  
  
#Write the vertical lines  
file.write('// Verticle lines \n')  
for v in vlines:  
    file.write('Line('+str(np.int(v[0]))+') =  
{'+str(np.int(v[1]))+', '+str(np.int(v[2]))+'};\n')  
  
#Write the horizontal lines  
file.write('// Horizontal lines \n')  
for h in hlines:  
    file.write('Line('+str(np.int(h[0]))+') =  
{'+str(np.int(h[1]))+', '+str(np.int(h[2]))+'};\n')  
  
#Write the line loops  
surfacecounter = 1  
file.write('// Line loops \n')  
for i, L in enumerate(loop):  
  
    if i != len(loop)-2:  
        if np.mod(i+1, 3) == 0:  
            file.write('Line Loop('+str(np.int(L[0]))+') =  
{'+str(np.int(L[1]))+', '+str(np.int(L[2]))+', '+str(np.int(-L[3]))+', '+str(np.int(-L[4]  
)+'};\n')  
  
            if np.mod(i+2, 3) == 0:  
                file.write('Line Loop('+str(np.int(L[0]))+') =  
{'+str(np.int(L[1]))+', '+str(-np.int(L[2]))+', '+str(np.int(-L[3]))+', '+str(np.int(L[4]  
)+'};\n')  
  
            if np.mod(i+3, 3) == 0:  
                file.write('Line Loop('+str(np.int(L[0]))+') =  
{'+str(np.int(L[1]))+', '+str(np.int(L[2]))+', '+str(np.int(-L[3]))+', '+str(np.int(-L[4]  
)+'};\n')  
            #Handle the final loop...  
        else:
```

```
        file.write('Line Loop('+str(np.int(L[0]))+') =
{'+str(np.int(L[1]))+', '+str(-np.int(L[2]))+', '+str(np.int(-L[3]))+', '+str(np.int(L[4]
))+'};\n')

        file.write('Ruled Surface('+str(surfacecounter)+') =
{'+str(surfacecounter)+'};\n')
        surfacecounter = surfacecounter +1

#Set up sidel, side2, and the outer wall
#Side1
string = ''
file.write('Physical Surface("Left") = {')
for val in np.arange(loop[-1][0])+1:
    if np.mod(val, 3) == 0:
        string = string+(np.str(np.int(val))+',')
file.write(string[:-1]+'};\n')

#Side2
string = ''
file.write('Physical Surface("Right") = {')
for val in np.arange(loop[-1][0])+1:
    if np.mod(val+1, 3) == 0:
        string = string+(np.str(np.int(val))+',')
file.write(string[:-1]+'};\n')

#outerwall
string = ''
file.write('Physical Surface("Outerwall") = {')
for val in np.arange(loop[-1][0])+1:
    if np.mod(val+2, 3) == 0:
        string = string+(np.str(np.int(val))+',')
file.write(string[:-1]+'};\n')

#Set up line loop and empty surface for axis:
string = ''
file.write('Physical Line("Axis") = {')
for val in np.arange(length-1)+(length)*2-1:
    string = string+(np.str(np.int(val))+',')
file.write(string[:-1]+'};\n')

#Form a volume for each cell in the center
#Write the internal line
interiorloopstart = surfacecounter
for val in np.arange(length):
    string = 'Line loop('+str(surfacecounter)+') = {'
    baseval = (val+length-1)*3
    string = string +
str(-int(baseval+3))+', '+str(int(baseval+2))+', '+str(-int(baseval+1))
```

```
file.write(string+'};\n')
file.write('Ruled Surface('+str(surfacecounter)+'') =
{' +str(surfacecounter)+'};\n')

surfacecounter = surfacecounter+1

file.write('Physical Surface("Input") = {' +str(interiorloopstart)+'};\n')
file.write('Physical Surface("Output") = {' +str(surfacecounter-1)+'};\n')

volcounter = 1
for val in (np.arange(length-1)):

    string = 'Surface loop('+str(volcounter)+'') = {'
    string = string+(np.str(np.int(val*3+1))+' ,')
    string = string+(np.str(np.int(val*3+2))+' ,')
    string = string+(np.str(np.int(val*3+3))+' ,')
    string = string+(np.str(np.int(interiorloopstart+val))+' ,')
    string = string+(np.str(np.int(interiorloopstart+val+1)))

    file.write(string+'};\n')
    file.write('Volume('+str(volcounter)+'') = {' +str(volcounter)+'};\n')
    volcounter = volcounter+1

#Set up the mesh structure for the horizontal lines

string = ''
file.write('Transfinite Line {'')
for val in np.arange(nhorz):
    #if val != nhorz-1:
    if np.mod(val, 3) != 0:
        string = string+str(int(val+nvert+1))+' , '

file.write(string[:-1]+'} = '+str(hnum)+' Using Progression '+str(hprog)+';\n')

#Set up the mesh for the outer wall
string = ''
file.write('Transfinite Line {'')
for val in np.arange(nhorz):
    #if val != nhorz-1:
    if np.mod(val, 3) == 0:
        string = string+str(int(val+nvert+1))+' , '

file.write(string[:-1])
file.write('} = 1;\n')

#Set up the mesh structure for the vertical lines
file.write('Transfinite Line {'')
for val in np.arange(nvert):
    if val != nvert-1:
```

```
    #if np.mod(val, 3) != 0 and val != nhorz-1:
        file.write(str(int(val+1))+',')
    if val == nvert-1:
        #if (np.mod(val, 3) != 0) and val == nhorz-1:
            file.write(str(int(val+1)))
file.write('} = 1;\n')

file.write('Transfinite Surface "*";\n')
file.write('Recombine Surface "*";\n')

file.write('Transfinite Volume "*";\n')

string = 'Physical Volume("internal") = {'

for val in np.arange(volcounter-1)+1:
    string = string+str(int(val))+','

file.write(string[:-1]+'};\n')

file.close()
```

```
/*-----*- C++ -*-----*/
|=====|
| \ / Field | OpenFOAM: The Open Source CFD Toolbox |
| \ / Operation | Version: 2.4.0 |
| \ / And | Web: www.OpenFOAM.org |
| \ / Manipulation |
|-----*/
FoamFile
{
    version      2.0;
    format       ascii;
    class        dictionary;
    location     "system";
    object       fvSchemes;
}
// *****

fluxScheme      Kurganov;

ddtSchemes
{
    default      Euler;
}

gradSchemes
{
    default      Gauss linear;
}

divSchemes
{
    default      none;
    div(tauMC)   Gauss linear;
}

laplacianSchemes
{
    default      Gauss linear corrected;
}

interpolationSchemes
{
    default      linear;
    reconstruct(rho) vanLeer;
    reconstruct(U) vanLeerV;
    reconstruct(T) vanLeer;
}

snGradSchemes
{
```

```
    default      corrected;  
}
```

```
// *****  
//
```



```
/*-----*- C++ *-----*/
|=====|
| \ / Field | OpenFOAM: The Open Source CFD Toolbox |
| \ / Operation | Version: 2.4.0 |
| \ / And | Web: www.OpenFOAM.org |
| \ / Manipulation |
|-----*/
FoamFile
{
    version      2.0;
    format       ascii;
    class        dictionary;
    location     "system";
    object       fvSolution;
}
// *****

solvers
{
    "(rho|rhoU|rhoE)"
    {
        solver      diagonal;
    }

    U
    {
        solver      smoothSolver;
        smoother    GaussSeidel;
        nSweeps      2;
        tolerance    1e-10;
        relTol       0;
    }

    e
    {
        $U;
        tolerance    1e-10;
        relTol       0;
    }
}

// *****
```

```
/*-----*- C++ -*-----*/
|=====|
| \ / Field | OpenFOAM: The Open Source CFD Toolbox |
| \ / Operation | Version: 2.4.0 |
| \ / And | Web: www.OpenFOAM.org |
| \ / Manipulation |
|-----*/
FoamFile
{
    version      2.0;
    format       ascii;
    class        dictionary;
    location     "constant";
    object       thermophysicalProperties;
}
// *****

thermoType
{
    type         hePsiThermo;
    mixture      pureMixture;
    transport    sutherland;
    thermo       hConst;
    equationOfState perfectGas;
    specie       specie;
    energy       sensibleInternalEnergy;
}

mixture
{
    specie
    {
        nMoles      1;
        molWeight    28.96;
    }
    thermodynamics
    {
        Cp          1004.5;
        Hf          0;
    }
    transport
    {
        As          1.458e-06;
        Ts          110.4;
        Pr          1;
    }
}

// *****
```



```
/*-----*- C++ -*-----*/
|=====|
| \ \ / F i e l d | OpenFOAM: The Open Source CFD Toolbox |
| \ \ / O p e r a t i o n | Version: 2.4.0 |
| \ \ / A n d | Web: www.OpenFOAM.org |
| \ \ / M a n i p u l a t i o n |
|-----*/
FoamFile
{
    version      2.0;
    format       ascii;
    class        dictionary;
    location     "constant";
    object       turbulenceProperties;
}
// ***** //

simulationType laminar;

// ***** //
```

```
/*-----*- C++ -*-----*/
|=====|
| \ \ / F i e l d | OpenFOAM: The Open Source CFD Toolbox |
| \ \ / O p e r a t i o n | Version: 2.4.0 |
| \ \ / A n d | Web: www.OpenFOAM.org |
| \ \ / M a n i p u l a t i o n |
|-----*/
FoamFile
{
    version      2.0;
    format       ascii;
    class        polyBoundaryMesh;
    location     "constant/polyMesh";
    object       boundary;
}
// *****

5
(
    Input
    {
        type           patch;
        physicalType   patch;
        nFaces         199;
        startFace      158204;
    }
    Output
    {
        type           patch;
        physicalType   patch;
        nFaces         199;
        startFace      158403;
    }
    Outerwall
    {
        type           patch;
        physicalType   patch;
        nFaces         399;
        startFace      158602;
    }
    Right
    {
        type           wedge;
        physicalType   wedge;
        nFaces         79401;
        startFace      159001;
    }
    Left
    {
        type           wedge;
    }
}
```

```
    physicalType    wedge;  
    nFaces          79401;  
    startFace       238402;  
  }  
)  
  
// *****  
//
```

```
/*-----* C++ *-----*/
|=====|
| \ / Field | OpenFOAM: The Open Source CFD Toolbox |
| \ / Operation | Version: 2.4.0 |
| \ / And | Web: www.OpenFOAM.org |
| \ / Manipulation |
|-----*/
FoamFile
{
    version      2.0;
    format       ascii;
    class        volScalarField;
    object       p;
}
// *****

dimensions      [1 -1 -2 0 0 0];

internalField   uniform 15000;

boundaryField
{

    Output
    {
        type waveTransmissive;
        field p;
        phi phi;
        rho rho;
        psi thermo:psi;
            1.4;
        fieldInf 2533.33;
        lInf 1;
        value uniform 2533.33;
    }

    Outerwall
    {
        type      slip;
    }

    Input
    {
        type      totalPressure;
        U          U;
        phi        phi;
        rho        none;
    }
}
```

```
psi          none;
             1.4;
p0           uniform 101325;
value       uniform 101325;

}

Left {type wedge;}
Right {type wedge;}
Axis {type empty;}
}

// ***** //
```



```
/*-----*- C++ -*-----*/
|=====|
| \ / Field | OpenFOAM: The Open Source CFD Toolbox |
| \ / Operation | Version: 2.4.0 |
| \ / And | Web: www.OpenFOAM.org |
| \ / Manipulation |
|-----*/
FoamFile
{
    version      2.0;
    format       ascii;
    class        volVectorField;
    object       U;
}
// *****

dimensions      [0 1 -1 0 0 0];

internalField   uniform (5 0 0);

boundaryField
{
    Input
    {
        type      zeroGradient;
    }

    Output
    {
        type      zeroGradient;
    }

    Outerwall
    {
        type      slip;
    }

    Left {type wedge;}
    Right {type wedge;}
    Axis {type empty;}
}
// *****
```

```
/*-----* C++ *-----*/
|=====|
| \ / Field | OpenFOAM: The Open Source CFD Toolbox |
| \ / Operation | Version: 2.4.0 |
| \ / And | Web: www.OpenFOAM.org |
| \ / Manipulation |
|-----*/
FoamFile
{
    version      2.0;
    format       ascii;
    class        volScalarField;
    object       T;
}
// *****

dimensions      [0 0 0 1 0 0 0];

internalField   uniform 298.0;

boundaryField
{
    Input
    {
        type      fixedValue;
        value     uniform 298.0;
    }

    Output
    {
        type      zeroGradient;
    }

    Outerwall
    {
        type      slip;
    }

    Left {type wedge;}
    Right {type wedge;}
    Axis {type empty;}
}
// *****
```

```
/*-----*- C++ -*-----*/
|=====|
| \ / Field | OpenFOAM: The Open Source CFD Toolbox |
| \ / Operation | Version: 2.4.0 |
| \ / And | Web: www.OpenFOAM.org |
| \ / Manipulation |
|-----*/
FoamFile
{
    version      2.0;
    format       ascii;
    class        dictionary;
    location     "system";
    object       controlDict;
}
// *****

application    rhoCentralFoam;

startFrom      latestTime;

startTime      0.0161;

stopAt         endTime;

endTime        2e-02;

deltaT         1e-10;

writeControl   adjustableRunTime;

writeInterval  1e-04;

cycleWrite     0;

writeFormat    ascii;

writePrecision 15;

writeCompression off;

timeFormat     general;

timePrecision  6;

adjustTimeStep yes;

maxCo          0.5;

maxDeltaT      1;
```

// \*\*\*\*\* //  
// \*\*\*\*\* //

```
/*-----*- C++ -*-----*/
|=====|
| \ / Field | OpenFOAM: The Open Source CFD Toolbox |
| \ / Operation | Version: 2.4.0 |
| \ / And | Web: www.OpenFOAM.org |
| \ / Manipulation |
|-----*/
FoamFile
{
    version      2.0;
    format       ascii;
    class        dictionary;
    location     "system";
    object       sampleDict;
}
// *****

interpolationScheme cellPoint;

setFormat      raw;

sets
(
    horizontalLine
    {
        type      uniform;
        axis      distance;
        start     ( 0.1 0.0 0.005 );
        end       ( 0.1 0.0 1.995 );
        nPoints   100;
    }
);

fields        ( p U T);

// *****
```